# Study of heat transfer parameters on rhodium target for <sup>103</sup>Pd production

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**Abstract.** The efficiency of cooling and the particle beam characteristics are important when high beam current irradiations are intended for production of radionuclides. The efficiency of cooling is determined by both the target carrier geometry and the flow rate of coolant, while the beam characteristics deal with the current density distribution on the irradiated surface area. Heat transfer on rhodium target to produce <sup>103</sup>Pd via the <sup>103</sup>Rh(p,n)<sup>103</sup>Pd reaction was investigated and the beam current was obtained more than 500  $\mu$ A.

**Key words:** heat transfer • <sup>103</sup>Pd • production • rhodium • target

# Introduction

The suitability of a given radionuclide for brachytherapy is determined by its half-life and by the type, energy and abundance of the emitted radiation. Due to the <sup>103</sup>Pd half-life (16.97 days) and its electron capture decay resulting in the abundant emission of Auger electrons and low-energy X-rays [EC,  $K_{\alpha} = 20.1$  keV (64.7%),  $K_{\beta} = 22.7$  keV (12.3%)], <sup>103</sup>Pd can be used for the preparation of seeds as permanent interstitial implants for the treatment of rapidly proliferating tumours. Along with <sup>125</sup>I, the radionuclide has been used in the treatment of various cancers such as eye, brain, neck, uterus, colon, but it is now primarily used for prostate tumours. Over the last decade, it has proven to be very effective in treating this cancer with minimum side effects [9–11].

For cyclotron production of  $^{103}$ Pd, the most suitable is the reaction  $^{103}$ Rh(p,n) $^{103}$ Pd [2, 6, 12]. One of the most important parameters in producing radionuclides by target bombardment with energetic charged particles is the heat transfer factor. When the accelerated particles hit atoms at the target, there is an interaction that leads to excitation, ionization and Bremsstrahlung. Some of the kinetic energy converts into heat that produces undesirable effects on the production conditions (such as melting of the target and peeling of the electroplating layer). In this research, the heat transfer factor on rhodium target for producing  $^{103}$ Pd has been studied.

#### **Methods**

The production of <sup>103</sup>Pd is carried out via the <sup>103</sup>Rh(p,n)<sup>103</sup>Pd reaction which is well suited to lowenergy cyclotrons [2, 6, 12]. Cyclone-30 (IBA, Belgium,

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Received: 9 January 2009 Accepted: 11 May 2009  $I_{\text{max}} = 350 \ \mu\text{A}, E_{P_{\text{max}}} = 30 \ \text{MeV}$ ) at the Agricultural, Medical and Industrial Research School was employed. The solid targetry system in this cyclotron is made up of a pure copper backing on which the target materials are electrodeposited (surface area,  $s = 11.69 \ \text{cm}^2$ ).

To take full benefit of the excitation function for the <sup>103</sup>Rh(p,n)<sup>103</sup>Pd reaction and to avoid the formation of the radionuclide <sup>101</sup>Pd impurity, the proton entrance energy should be  $E_{P1} = 18$  MeV [2, 6, 12]. The physical thickness of the rhodium layer is chosen in such a way that for a given beam/target angle geometry the particle exit energy should be  $E_{P2} = 5$  MeV. According to SRIM code [15], the thickness has to be 475 µm for 90° geometry. To minimize the thickness of the rhodium layer a 6° geometry is preferred, in which case a 48 µm layer is recommended.

The target that undergoes bombardments by the proton beam at the cyclotron production consists of three layers, namely: a) rhodium layer, b) copper layer and c) copper layer without induced proton beam (see Fig. 1).

This target is surrounded by the accelerator vacuum on one side and by the coolant fluid on the other side. The rhodium target was irradiated with a proton beam at an incidence angle of 6°, and some of the proton energy converts into heat. If the irradiation is considered uniform on the rhodium layer, it means a current density  $(\mu A \cdot mm^{-2})$  on this layer and the relationship between the heat production rate  $Q_{Rh}$  (W) and the heat production rate per volume unit  $Q'''_{Rh}$  (W·cm<sup>-3</sup>) will be:

(1) 
$$Q_{\rm Rh} = Q^{\rm m}_{\rm Rh} \times a \times s$$

In this relation, *s* is the target surface area in cm<sup>2</sup>. The exiting protons from the rhodium layer (*a* layer in cm) are stopped completely in the second layer (cuprous layer, *b* in cm). The relation between heat production rate in this layer  $Q_{Cu}$  and the heat production rate per volume unit  $Q''_{Cu}$  is as follows:

(2) 
$$Q_{\rm Cu} = Q^{\rm m}{}_{\rm Cu} \times b \times s$$

The third layer (c) serves just as a mechanic retentive device. The heat produced in the first and second layers, transfers to the coolant fluid through this last layer in the form of conduction.

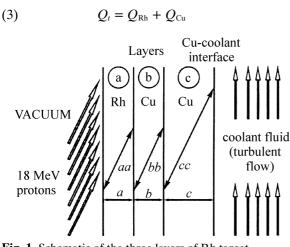


Fig. 1. Schematic of the three layers of Rh target.

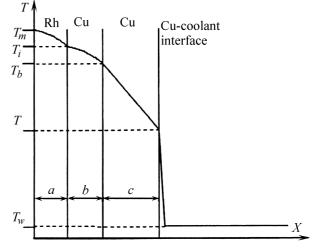


Fig. 2. Profile of temperature in the Rh target.

Under stationary conditions, heat transfer appears as a temperature profile (Fig. 2) in which  $T_m$  is the temperature at the vacuum-rhodium interface,  $T_i$  is the rhodium and second layer interface temperature, and  $T_b$  is the temperature of the second and third layer interface. Finally,  $T_c$  is the temperature of the Cu-coolant interface.  $T_w$  is the temperature of the coolant fluid.

The temperature profile can be obtained solving the Poisson, Laplace and Newton's equations in the three layers and the coolant. Poisson's equation in the first and second layer is  $d^2T/dx^2 = -Q''/k$  where k (W·cm<sup>-1</sup>·K<sup>-1</sup>) is the metal thermal conductivity. The Laplace's equation in the third layer is  $d^2T/dx^2 = 0$  and Newton's equation for temperature profile in the third layer (c) and the coolant interface is  $Q_t = h \cdot s \cdot \Delta T$  [1, 7, 8, 14]. In the latter formula, h is the heat transfer coefficient in convection method (W·cm<sup>-2</sup>·K<sup>-1</sup>).

Integration of Poisson, Laplace's equations and appropriate boundary conditions will result in a temperature profile relation which gives  $Q_{\text{Rh}}$ ,  $Q_{\text{Cu}}$ ,  $Q_t$  for beam current (*I*) and proton energy ( $\Delta E$ ) in the related layers.

The mentioned temperature profile relation is:

(4) 
$$T_{m} - T_{w} = \left(\frac{Q_{Rh}}{k_{Rh}} \cdot \frac{a}{s}\right) + \left(\frac{Q_{Rh}}{k_{Cu}} \cdot \frac{b}{s} + \frac{Q_{Cu}}{2k_{Cu}} \cdot \frac{b}{s}\right) + \left(\frac{Q_{T}}{k_{Cu}} \cdot \frac{c}{s}\right) + \left(\frac{Q_{T}}{h \cdot s}\right)$$

 $k_{\rm Rh}$  and  $k_{\rm Cu}$  (W·cm<sup>-1</sup>·K<sup>-1</sup>) are the rhodium and cupper thermal conductivity, respectively. The heat produced in the rhodium layer can be obtained by considering the mean energy of the proton and the total number of protons that enter in this layer, in units of time:

(5) 
$$Q_{\rm Rh} = N \cdot \Delta E_P (W)$$

 $\Delta E_P$  is the beam energy variation at the desired layer and N is the total number of protons entering the layer per second, and it is obtained from the following equation:

(6) 
$$N = \frac{I \times 10^{-6} (C/s)}{q_p(C)} = 6.25 \times 10^{12} \times I(s^{-1})$$

where: I(A) and  $q_p(C)$  are the beam current and the proton charge, respectively.

After the protons exit from the rhodium layer, they will stop in the second layer (*b* layer). In this layer, the heat production rate ( $Q_{Cu}$ ) can be calculated as well. By assuming turbulent flow regime, it means no boiling will happens in the interface between the third layer and the coolant, the difference between  $T_c$  and  $T_w$  can be obtained from the following equation:

(7) 
$$Q_t = h \cdot s \cdot (T_c - T_w)$$

Dittus and Boelter proposed the following equation to predict heat transfer coefficient for turbulent flow of fluid [3–5, 13]:

(8) 
$$h = C \times \frac{k_c}{D_e} \times R_e^m \times P_r^n$$

where: *n*, *m*, *C* are constants,  $R_e$  is Reynolds number dimensionless,  $P_r$  is Prandtl number,  $D_e$  is the equivalent diameter of the coolant channel (cm) and finally  $k_c$  is the fluid thermal conductivity (W·cm<sup>-1</sup>·K<sup>-1</sup>).

 $R_e$  defined by  $R_e = D_e \cdot v \cdot \rho / \mu$ , where v is the average linear velocity of coolant fluid (cm·s<sup>-1</sup>),  $\rho$  is the coolant density and  $\mu = 10^{-2}$  is the kinematic viscosity of the coolant fluid in cm<sup>2</sup>·s<sup>-1</sup>. The quantity  $D_e$  is obtained in the following equation:

$$(9) D_e = 4 \times O/L$$

where *O* is the cross-sectional area of the coolant chan-**Table 1.** Nomenclature and values of parameters

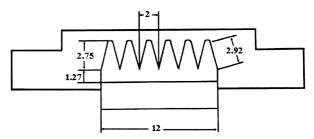


Fig. 3. Cross-section of the Rh target.

nel and *L* is its perimeter. Figure 3 shows the crosssectional area of the coolant channel. By choosing the right parameters in order to have  $R_e > 10,000$  condition, turbulent flow condition will be satisfy. The magnitude of *h* depends on  $P_r$  too, which is shown below, as an equation,  $k_{\rm H2O}$  (W·cm<sup>-1</sup>·K<sup>-1</sup>) and  $C_p$  (J·cm<sup>-3</sup>·K<sup>-1</sup>) are the thermal conductivity and specific heat of the coolant fluid, respectively.

(10) 
$$P_{\gamma} = C_p \times \mu / k_{\rm H_{2O}}$$

# **Results and discussion**

In Table 1 are shown all the data of target and parameters that were considered for the present work. The thermal shock tests involved the heating of the target up to 500°C (the temperature that the Rh layer can experience during a high current irradiation) for 1 h followed by submersion of the hot target in cold water

	Input data			
Quantity	Symbol	Value	Dimension	
Thickness of the third layer	С	0.1	cm	
Reynolds number	$R_{e}$	67251	dimensionless	
Empirical constant	al constant m		dimensionless	
Empirical constant	п		dimensionless	
Empirical constant	С		dimensionless	
Coolant flow rate	f		$cm^3 \cdot s^{-1}$	
Prandtl number			dimensionless	
Rhodium thermal conductivity $k_{\rm Rh}$		1.50	$(W \cdot cm^{-1} \cdot K^{-1})$	
Cupper thermal conductivity $k_{Cu}$		4.00	$(W \cdot cm^{-1} \cdot K^{-1})$	
Thermal conductivity of the coolant fluid	$k_{ m H_{2O}}$	0.597 E-2	$(W \cdot cm^{-1} \cdot K^{-1})$	
Specific heat of the coolant fluid	$\overline{C_p}$	4.18	$(J \cdot cm^{-3} \cdot K^{-1})$	
Average linear velocity of coolant	ν	2627	$\text{cm} \cdot \text{s}^{-1}$	
Kinematic viscosity of the coolant fluid	μ	1 E-2	$cm^2 \cdot s^{-1}$	
Temperature at the vacuum-rhodium interface	$T_m$	500	°C	
Temperature of the coolant fluid	$T_w$	20	°C	
	Calculated values			
Quantity	Symbol	Value	Dimension	
Thickness of the Rh layer	а	48 E-4	cm	
Thickness of the second layer	b	8 E-4	cm	
Cross-sectional area of the coolant channel	0	0.317	$cm^2$	
Perimeter of the coolant channel	L	4.95	cm	
Hydraulic equivalent diameter	$D_e$	0.256	cm	
Heat transfer coefficient	h	8.5	$(W \cdot cm^{-2} \cdot K^{-1})$	
Heat production rate in the Rh layer $Q_{\rm Rh}$		13 I	(W)	
Heat production rate in the Cu layer $\tilde{Q}$		5 I	(W)	
Heat production rate	$Q_t$	18 I	(W)	

(20°C). Observation of neither crack formation nor peeling off the rhodium layers indicated a good adhesion for the purpose.

By taking into account all the concepts in previous section, Eq. (4) can be rewritten as follows:

$$Q_{\rm Rh} = (E_{P1} - E_{P2}) \cdot I = 13 I, \quad Q_{\rm Cu} = E_{P2} \cdot I = 5 I,$$
$$Q_t = E_{P1} \cdot I = 18 I$$

(11) 
$$T_m - T_w = \left(\frac{II}{2k_{Rh}} \cdot \frac{I}{s}\right) + \left(\frac{(E_{P1} - E_{P2}) \cdot I}{k_{Cu}} \cdot \frac{b}{s} + \frac{E_{P2} \cdot I}{2k_{Cu}} \cdot \frac{b}{s}\right) + \left(\frac{E_{P1} \cdot I}{k_{Cu}} \cdot \frac{c}{s}\right) + \left(\frac{E_{P1} \cdot I}{h \cdot s}\right)$$

where  $E_{P1}$  and  $E_{P2}$  are the entrance and outlet proton energy in the first layer, respectively,

(12) 
$$T_m - T_w = (T_m - T_i) + (T_i - T_b) + (T_b - T_c) + (T_c - T_w)$$

Replacing the parameters from Table 1 in Eq. (11), results in 2100 µÅ for a maximum proton current at 500°C temperature, which is the temperature of thermal shock test. Different quantities of temperature differences in layers have been obtained from the above equation and according to Table 2. The maximum tolerated current density for target equals:

(13) 
$$i_m = \frac{I_m}{s} = \frac{2100}{1169} = 1.796 \,\mu\text{A/mm}^2$$

Table 2. Calculated temperature decrements over Rh target

<i>Ι</i> (μΑ)	$T_m - T_i$	$T_i - T_b$	$T_b - T_c$	$T_c - T_w$	$T_m - T_w$
100	0.18	0.03	3.85	18.12	22.17
200	0.36	0.05	7.70	36.23	44.34
300	0.53	0.08	11.55	54.35	66.51
400	0.71	0.11	15.40	72.46	88.68
500	0.89	0.13	19.25	90.58	110.84
600	1.07	0.16	23.10	108.69	133.01
700	1.25	0.19	26.95	126.81	155.18
800	1.42	0.21	30.80	144.92	177.35
900	1.60	0.24	34.64	163.04	199.52
1000	1.78	0.27	38.49	181.15	221.69
1100	1.96	0.29	42.34	199.27	243.86
1200	2.14	0.32	46.19	217.38	266.03
1300	2.31	0.34	50.04	235.50	288.20
1400	2.49	0.37	53.89	253.61	310.36
1500	2.67	0.40	57.74	271.73	332.53
1600	2.85	0.42	61.59	289.84	354.70
1700	3.02	0.45	65.44	307.96	376.87
1800	3.20	0.48	69.29	326.07	399.04
1900	3.38	0.50	73.14	344.19	421.21
2000	3.56	0.53	76.99	362.30	443.38
2100	3.74	0.56	80.84	380.42	465.55
2200	3.914	0.58	84.69	398.53	487.72

#### The effective parameters on maximum current target

### Effect of rhodium carrier material

In a previous section, copper with 4.00 (W·cm<sup>-1</sup>·K<sup>-1</sup>) thermal conductivity was considered as rhodium carrier. If silver is used instead of copper, the maximum allowable target current will increase a little to  $k_{Ag}$  = 4.30 ( $W \cdot cm^{-1} \cdot K^{-1}$ ), but because of its high price, there is no advantage from the economical point of view.

#### Effect of coolant channel shape

Calculating heat transfer coefficient was done by the convection method in the case of coolant channel with fins. The heat transfer coefficients obtained were 5.14 and 8.04 W·cm<sup>-2</sup>·K<sup>-1</sup> by eliminating fins and decreasing the effective depth of channel to 2.5 mm with no fins, respectively. Comparison of the recent amount of h(= 8.5) with quantities of h for carrier with no fins, results in these cases, h decreases by 40% and 5%, respectively. Decreasing the heat transfer coefficient allows decreasing the beam current.

#### Effect of coolant flow rate

If the flow rate is increased (e.g.  $f = 1666 \text{ cm}^3 \cdot \text{s}^{-1}$ ), for the carrier with no fins:

(14) 
$$v = \frac{f}{O} = \frac{1666}{0.300} = 5553 \text{ cm/s}$$

(15) 
$$R_e = \frac{D_e \cdot v \cdot \rho}{\mu} = \frac{0.413 \times 5553 \times 1}{1 \times 10^{-2}} = 229,398$$

and Prandtl number is equal to 7, then the heat transfer coefficient will be 14  $W \cdot cm^{-2} \cdot K^{-1}$ . So, the beam current can be improved.

### Conclusion

High beam current ( $I > 500 \mu A$ ) irradiations can be used to produce the <sup>103</sup>Pd radionuclide from a Rh target. The relation Eq. (11) shows that decreasing the thickness of layer c has a positive effect on a proton current increase. The effect of channel shape in the case of with or without fins, influences the parameter h; increasing the heat transfer coefficient allows increasing the current. Also increasing the coolant flow rate has a positive effect on the parameter *h*.

## References

- 1. Bernath L (1960) A theory of local-boiling burnout and its application to existing data. Chem Eng Prog Symp Ser 56:95-116
- 2. Broeders CHM, Konobeyev AY, Korovin YA, Blann VPM (2006) Forschungszentrum Karlsruhe Report FZKA 7183, http://bibliothek.fzk.de/zb/berichte/FZKA7183.pdf
- 3. Chen JC (1966) A correlation for boiling heat transfer to saturated fluids in convective flow. Ins Eng Chem 5;3:322-329

(11)

- Dittus FW, Boelter LMK (1930) Heat transfer in automobile radiators of the tubular type. Univ Calif Publs Eng 2:443–446
- Gambill WR, Mochizuki T (1988) Advanced neutron source design: burnout heat flux correlation development. Trans Am Nucl Soc 57:298–300
- Hermanne A, Sonck M, Fenyvesi A, Daraban L (2000) Study on production of <sup>103</sup>Pd and characterization of possible contaminants in the proton irradiation of <sup>103</sup>Rh up to 28 MeV. Nucl Instrum Methods Phys Res B 170:281–299
- IAEA (2008) Cyclotron produced radionuclides: principles and practice. Technical Reports Series no 465. IAEA, Vienna, pp 96–129
- McAdams WH, Kennel WE, Minden CS, Carl R, Picornell P, Dew JE (1949) Heat transfer at high rates to water with surface boiling. Ins Eng Chem 41:1945–1953
- Nag S, Riborich M, Cai JZ (1996) Palladium-103 vs. iodine-125 brachytherapy in the danning-pap rat prostate tumor. Endocuriether Hyperthem Oncol 12:119–124
- Nag S, Sweeney PJ, Wientjes MG (1993) Dose-response study of iondine-125 and palladium-103 brachytherapy in a rat prostatic tumor. Endocuriether Hyperthem Oncol 9:97–104

- Porrazzo MD, Hilaris BS, Moorthy CR (1992) Permanent interstitial implantation using <sup>103</sup>palladium: the New York Medical College preliminary experience. Int J Radiat Oncol Biol Phys 23;5:1033–1036
- 12. Sudár S, Cserpak F, Qaim SM (2002) Measurements and nuclear model calculations on proton-induced reactions on <sup>103</sup>Rh up to 40 MeV: evaluation of the excitation function of the <sup>103</sup>Rh(p,n)<sup>103</sup>Pd reaction relevant to the production of the therapeutic radionuclide <sup>103</sup>Pd. Appl Radiat Isot 56:821–831
- Sudo Y, Miyata K, Ikawa H, Kaminaga M, Ohkawara M (1985) Experimental study of differences in DNB heat flux between upflow and downflow in vertical rectangular channel. J Nucl Sci Technol 22:604–618
- Tanaka F, Hibiki T, Saito Y, Takeda T, Mishima K (2001) Heat transfer study for thermal-hydraulic design of the solid-target of spallation neutron source. J Nucl Sci Technol 38:832–843
- Ziegler JF, Biersack JP, Littmark U (2006) The code of SRIM – the stopping and range of ions in matter. IBM Research, New York, USA, www.srim.org